Duct Leakage White Paper

Introduction
Air duct leakage should be a concern to both the design engineer and the building owner because of its potential impact on the initial construction costs, energy usage, and system performance.

Every duct system requires power. Power drives the fan or blower unit needed to move the air over or through the resistance elements and power operates the compressor or heating elements depending on the desired function. No matter the size, the total benefit of the duct system is based and measured on the peak fan efficiency for that system.

Background
Small commercial buildings typically use a packaged air handling unit where all of the supply and return air ductwork is in the conditioned space. If the design air flows are not properly delivered to the building HVAC loads, the occupants will respond with corresponding higher or lower temperature settings to meet their comfort requirements. The end result is higher energy costs.

In large complex buildings, particularly those with variable air volume systems (VAV), recent research has found that a more complex interaction occurs which requires computer simulation to quantify the energy impact. The Public Interest Energy Research Program (PIER) reported “… when conditioned air leaks from the supply ducts, the heating or cooling energy associated with leakage heats or cools the return air and changes its temperature (and enthalpy).”¹

Typically “…one third of the total annual energy consumption is related to HVAC (heating, ventilation, and cooling). In addition, 39% of this HVAC consumption is associated with fan operation.”²

Ductwork
The key elements influencing ductwork leakage start with the very basics – size, shape and construction materials. Ductwork is made from a wide range of materials -- galvanized steel, carbon steel, aluminum, stainless steel, fiberglass, polyvinyl chloride (PVC), polyvinyl steel (PVS), duct board, and others. Perhaps the most common material used is galvanized steel. Ductwork is available in rectangular, round, and flat oval geometric shapes. The particular shape that is selected for a specific system should adhere to minimizing the initial installed

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cost and annual operating costs, as well as conform to the constraints of the building envelope.

There are many reasons why SMACNA HVAC Systems Duct Design Manual recommends the following:
1. Use the minimum number of fittings,
2. Consider the use of semi-extended plenums,
3. Seal ductwork to minimize air leakage,
4. Consider using round duct, and
5. When using rectangular ductwork, maintain an aspect ration as close to 1-to-1 as possible.

There are several reasons for specifying round ductwork versus rectangular ductwork:
1. Lowest possible duct friction loss for a given perimeter,
2. Lowest weight based upon the same airflows, pressures, and friction loss rates
3. Requires less supports per running foot,
4. Handles negative pressures with less weight and reinforcement,
5. Handles higher air velocities than rectangular ductwork while achieving the same acoustic design criteria, and most importantly,
6. Least expensive to seal for air leakage.

**Ductwork Sealing**

The engineering community has traditionally specified SMACNA’s three distinct duct sealing classes (A, B, or C), which differ in their requirements for sealing the transverse joint, longitudinal seams, and duct penetrations. Typically the design engineer will specify a Seal Class and a percentage of the design airflow as an acceptable air leakage rate.

As reported in the SMACNA 1990 HVAC Systems Duct Design Manual, duct leakage previously specified as an arbitrarily established percentage of the airflow were impossible to attain by the installing contractor. Joint research conducted by SMACNA and ASHRAE has since developed a methodology used to relate the amount of ductwork leakage to the ductwork surface area and the design static pressure independent of the actual airflow in the ductwork

SMACNA publishes a table (Table 1) that correlates the “Seal Class” (A, B, or C) and the “Leakage Class” (typically 3, 6, 12, or 48). Obviously, this assumes a superior application of sealants to the ductwork system.

<table>
<thead>
<tr>
<th>Seal Class</th>
<th>A</th>
<th>B</th>
<th>C</th>
</tr>
</thead>
<tbody>
<tr>
<td>Leakage Class – Rectangular</td>
<td>24</td>
<td>12</td>
<td>6</td>
</tr>
<tr>
<td>Leakage Class – Round</td>
<td>12</td>
<td>6</td>
<td>3</td>
</tr>
</tbody>
</table>

Table 1
ASHRAE’s Energy Standard 90.1 recognizes the Leakage Class method for

\[ L_{\text{max}} = C_L P^{0.65} \]

where:

- \( L_{\text{max}} \): maximum permitted leakage in cfm/100 ft\(^2\) ductwork surface area;
- \( C_L \): duct leakage class, cfm/100ft\(^2\) at 1 inch water gage;
- \( P \): test pressure, which shall be equal to the design duct pressure class rating in inches water column.

In addition, the leakage class recognizes that under the best conditions, rectangular ductwork will leak air at a rate twice greater than round.

**Methods of Sealing Ductwork**

Various recognized methods of sealing ductwork also vary in degrees of cost, quality, visual appearance, and performance. Choices range from various types of flanges, to slip fit connections that require liquids, mastics, tapes or heat-applied materials to seal the joints. In addition, several ductwork manufacturers offer a factory applied gasket with self-sealing characteristics that do not require the field application of external sealants. Exposed ductwork remains in vogue with the architectural tastes of both designers and building owners, a trend that requires more attention be paid to the ductwork sealing methods than in the past because of the visual esthetics.

**Energy Costs**

A typical response to unanticipated ductwork air leakage has:

1. Increased design airflows which increases the initial construction costs for equipment and ductwork,
2. Increased fan energy,
3. Increased energy for heating, cooling, and dehumidifying the air stream,
4. Increased difficulty in air balancing the system airflows,
5. Impacts on the indoor air quality (IAQ), and most importantly
6. Compromised occupant comfort.

Computer simulations previously reported through PIER\(^1\) and confirmed by actual field measurements found the impact of air duct leakage on the total energy used to condition the occupied space:

- “…the increase in total annual HVAC site energy is 2% – 14%” and
- “…includes supply and return fan electricity consumption, chiller and cooling tower electricity consumption, boiler electricity consumption, and boiler natural gas consumption.”

For a less complex constant volume system, the annual increase in energy consumption for the supply fan alone (Table 2) ranges from 4% for a leakage class 6 system to 101% for a leakage class 48. Leakage class 3 is the tightest leakage class currently recognized by SMACNA and ASHRAE. A Leakage Class 48 is what one may expect with unsealed rectangular ductwork and is the highest recognized leakage class. This table represents what would be expected in an actual operating system based upon: changes in the system delivered cfm (CFM\(_2\)) due to air leakage; revised system total static pressure (TSP\(_2\)) as
calculated using established fan laws; and the actual required fan brake horsepower (BHP₂) using the published fan performance data from a nationally recognized fan manufacturer.

<table>
<thead>
<tr>
<th>Leakage Class</th>
<th>CFM Leakage</th>
<th>CFM₂ (iwg)</th>
<th>ISP₂ (iwg)</th>
<th>ESP₂ (iwg)</th>
<th>TSP₂ (iwg)</th>
<th>BHP₂</th>
<th>Total Dollars per Year</th>
<th>% Increase from Class 3 per Year</th>
</tr>
</thead>
<tbody>
<tr>
<td>3</td>
<td>143</td>
<td>20,190</td>
<td>2.04</td>
<td>2.04</td>
<td>4.08</td>
<td>16.70</td>
<td>$8,465</td>
<td>0%</td>
</tr>
<tr>
<td>6</td>
<td>291</td>
<td>20,482</td>
<td>2.10</td>
<td>2.10</td>
<td>4.19</td>
<td>17.40</td>
<td>$8,820</td>
<td>4%</td>
</tr>
<tr>
<td>12</td>
<td>605</td>
<td>21,087</td>
<td>2.22</td>
<td>2.22</td>
<td>4.45</td>
<td>19.03</td>
<td>$9,646</td>
<td>14%</td>
</tr>
<tr>
<td>24</td>
<td>1,309</td>
<td>22,395</td>
<td>2.51</td>
<td>2.51</td>
<td>5.02</td>
<td>22.79</td>
<td>$11,552</td>
<td>36%</td>
</tr>
<tr>
<td>48</td>
<td>3,098</td>
<td>25,493</td>
<td>3.25</td>
<td>3.25</td>
<td>6.50</td>
<td>33.60</td>
<td>$17,031</td>
<td>101%</td>
</tr>
</tbody>
</table>

Table 2

Assumptions:
- 20,000 design cfm
- 2.00 iwg ISP₁ (internal static pressure loss: air handler cabinet, filters, heating coil, and cooling coil)
- 2.00 iwg ESP₁ (external static pressure or friction losses in ductwork)
- 4.00 iwg TSP₁ (total static pressure or TSP₁ = ISP₁ + ESP₁)
- 3,000 ft² exposed ductwork
- Greenheck 44-AFSW-21 fan and Greenheck’s CAPS program (ver 2.6.2.1)
- 90% motor efficiency, 70% run time, $0.10 per KWh, PF=1.0

Conclusions

HVAC systems account for upwards of 40% of a building's annual energy usage. High efficiency equipment is one way to reduce the annual utility costs; however, understanding the different styles of duct systems and the impact on system leakage is key to realizing maximum system efficiency.

References

3. SMACNA HVAC Duct Construction Standards
4. SMACNA HVAC Systems Duct Design
5. ASHRAE 90.1
6. ASHRAE 2003 HVAC Applications Chapter 17 Sound and Vibration Control